

TN 4780 to re-sit spring 2014

25 June 2014

1. Since T stays uniform at all times we can use a macro balance.

$$-(2\pi RL + \pi R^2)h(T - T_a) = \pi R^2 L \rho C_p \frac{dT}{dt}$$

$$\frac{dT}{T - T_a} = \frac{d(T - T_a)}{T - T_a} = \frac{-(2\pi RL + \pi R^2)h}{\pi R^2 L \rho C_p} dt \equiv -K$$

$$\ln(T - T_a) = -Kt + C; \text{ at } t=0, T=T_0; C = \ln(T_0 - T_a)$$

$$\left(\frac{T - T_a}{T_0 - T_a}\right) = \exp(-Kt)$$

$$K = \frac{(2\pi(0.025)(0.3) + 2\pi(0.025)^2)10}{\pi(0.025)^2(12.3)900(2500)} = \frac{(0.0471 + 0.0039)10}{(0.000589)(900)(2500)}$$
$$= 0.000385$$

$$\frac{25 - 35}{0 - 35} = \exp(-0.000385t); t = 5054 \text{ s}$$

(abt 84 minutes)

2. This is unsteady conduction. BSL Fig 12.1-3 applies *

$$\alpha = k/\rho C_p = \frac{0.25}{900 \cdot 2500} = 1.11 \cdot 10^{-7}$$

$$\text{We want } \frac{T - T_0}{T_1 - T_0} = \frac{25 - 0}{35 - 0} = \frac{5}{7} = 0.714 \text{ at } r=0.$$

$$\text{From chart, } \frac{\alpha t}{R^2} \approx 0.3 = 1.11 \cdot 10^{-7} t / (0.025)^2$$

(hard to interpret)

$$t = 1690 \text{ s (about 25 min.)}$$

3. These two processes (convective heat transfer to surface, internal conduction) are in series. The slower answer (5054 s, question 1) is the better answer.

4. This is a process of superposition. The first change sets dimensionless T : $\frac{T - 50}{100 - 50}$. The second change is (-2) units in this dimensionless T . Therefore

$$\frac{T - 50}{100 - 50} = (\text{value of chart for 30 hr}) - 2 (\text{value of chart for 9 hr})$$

The chart is for unsteady conduction outwards from cylinder, from Carslaw + Jaeger.

*strictly, problem 2 involves the product method, but one dimension is much larger than the other. You can confirm for yourself that conduction in the z direction is unimportant.

$$\alpha = \frac{k}{\rho c_p} = \frac{0.831}{2008 \cdot 726} = 5.70 \cdot 10^{-7}$$

$$\text{at } 30 \text{ hr, } \frac{\alpha t}{R^2} = (5.70 \cdot 10^{-7})(30 \cdot 3600) / (0.025)^2 = 98$$

$$\text{at } 9 \text{ hr, } \frac{\alpha t}{R^2} = 29.5$$

$$r/R = 25/2.5 = 10. \quad \text{Lu}(r/R) = 1.$$

$$\text{From the chart, } \frac{T - T_0}{T - T_{\text{well}}} \approx 0.1 \text{ for } \frac{\alpha t}{R^2} = 29.5$$

$$\approx 0.22 \quad \frac{\alpha t}{R^2} = 98$$

$$\frac{T - 50}{100 - 50} = (0.22) - 2(0.1) = 0.02$$

$$T \approx 51^\circ \text{C}$$

5. What is Re? D_h for slit is 2 * width = 0.04 m.

$$Re = \frac{D_h v \rho}{\mu} = \frac{(0.04)(1)800}{0.000106} = 3.02 \cdot 10^5 \text{ Highly turbulent, so}$$

hydraulic dia. approx. is OK.

$$\text{For highly turbulent flow, } Nu = \frac{h D_h}{k} = 0.026 Re^{0.8} Pr^{1/3}$$

$$Pr = \frac{c_p \mu}{k} = \frac{4851 \cdot 0.000106}{0.62} = 0.829$$

$$Nu = (0.026)(3.02 \cdot 10^5)^{0.8} (0.829)^{1/3}$$

$$= 591 = \frac{h(0.04)}{(0.62)} \rightarrow h = 9166 \frac{\text{W}}{\text{m}^2 \cdot \text{K}}$$

$$a) Q = h A \Delta T; \frac{Q}{\text{Area}} = (9166)(150) = 1.37 \cdot 10^6 \text{ W/m}^2$$

b) If the water cools but we assume the rock hold

its same T_j , "Eq. III" applies:

$$Nu = \text{Lu} \left(\frac{T_0 - T_{b1}}{T_0 - T_{bc}} \right) Re Pr \left(\frac{D_h}{4L} \right)$$

$$591 = \text{Lu} \left(\frac{100 - 350}{100 - 200} \right) (3.02 \cdot 10^5) (0.829) \left(\frac{0.04}{4L} \right)$$

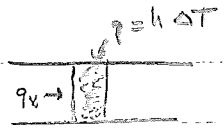
$$591 = \text{Lu} \left(\frac{250}{100} \right) 2504 \frac{1}{L}; \quad L = 3.88 \text{ m}$$

The water would flow only ~ 4 m. before cooling

to 200°C. (i.e., in 4 sec.)

In reality, the rock would heat + the ^{hot} fluid would flow further before cooling.

6. Because T does not vary w/ r , the control



volume can extend across the wire. Newton's

law at the surface enters the shell balance, as in the "coking fin" problem worked in class. Energy balance:

$$\pi R^2 q_x|_x - \pi R^2 q_x|_{x+\Delta x} - h 2\pi R \Delta x (T - T_a) + \pi R^2 \Delta x \dot{S} = 0$$

conduction in/out
Newton's law at surface
gen.
energy (zero at s.s.)

divide by $\pi R \Delta x$

$$\frac{R q_x|_x - R q_x|_{x+\Delta x}}{\Delta x} - h 2(T - T_a) + R \dot{S} = 0$$

let $\Delta x \rightarrow 0$

$$-R \frac{dq_x}{dx} - 2h(T - T_a) + R \dot{S} = 0$$

plug in Fourier's law: $q_x = -k \frac{dT}{dx}$

$$Rk \frac{dT}{dx} - 2h(T - T_a) + R \dot{S} = 0$$

BC: at $x=0$, $T=T_a$

$x=L$ $\frac{dT}{dx} = 0$ (reflecting $q_x \Rightarrow$ at $x=L$)